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UNIVERSITI TUN HUSSEIN ONN MALAYSIA

FINAL EXAMINATION SEMESTER 2 SESSION 2010/2011

NAME OF COURSE	:	DYNAMICS
COURSE CODE	•	BDA 20103 / BDA 2013
PROGRAM	:	BACHELOR OF MECHANICAL ENGINEERING WITH HONOURS
DATE OF EXAMINATION	:	APRIL / MAY 2011
DURATION	:	3 HOURS
INSTRUCTION	:	ANSWER ALL QUESTIONS.

THIS PAPER CONSISTS OF NINE (9) PAGES

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Q1 (a) Figure Q1(a) shows a diagram of a rigid body which is undergoing rotation about a fixed axis. Derive the equations which represent the magnitude of velocity and acceleration of point P by using the x-y coordinate system.

[5 marks]

- (b) A large window AB is closed by using the power window system through the application of cable as shown in Figure Q1(b). The window has the width of 3 m and the roller which connect the window and the wall is positioned at 1.5m from point A. If the cable is pulled by the force F with a constant speed of 0.8 m/s at the instant theta is equal to 55 degrees ($\theta = 55^\circ$);
 - (i) Determine the position coordinate equation which represents the motion of the system.
 - (ii) Find the distance x at the instant $\theta = 55^{\circ}$.
 - (iii) Calculate the angular velocity of the large window AB.
 - (iv) Find the angular acceleration of the large window AB.

[15 marks]

- Q2 (a) The link shown in Figure Q2 (a) is guided by two block A and B, which move in the fixed slots. The velocity of block A is 4 m/s downward.
 - (i) Draw kinematics diagram of the body.
 - (ii) Determine the magnitudes v_B and ω using the velocity equation of $v_B = v_A + v_{B/A}$ to points A and B.

[6 marks]

- (b) The connecting rod ABD is fixed at A and the length of rod AB and BD is 0.8 m respectively. Block D shown in Figure Q2 (b) moves with a speed of 6 m/s.
 - (i) Show the location of the instantaneous center (IC) on the kinematics diagram of link BD.
 - (ii) Determine the angular velocity of link BD.
 - (iii) Determine the velocity of B.
 - (iv) Draw the kinematics diagram of link AB.
 - (v) Determine the angular velocity of AB.

[14 marks]

Q3 Figure Q3 shows that pin A is fixed on bar DC. At the position shown, the bar DC is rotating counterclockwise at the constant rate, ω_{DC} = 2 rad/s. Pin A can only slide along the slotted plate EBO. Assume P be a point on EBO coincident with A. At this instant,
(a) Determine the angular velocity of EBO, ω_{EBO} rad/s.

[7marks]

(b) Determine the angular acceleration of EBO, $\alpha_{\text{EBO}} \operatorname{rad/s}^2$.

[8 marks]

(c) Draw the velocity and acceleration diagrams

[5 marks]

- Q4 A plate structure as seen in Figure Q4 has a pin rotation at O and simply supported at A. This plate structure is made of material with the density of 7800kg/m³. The thickness of the plate is 5mm.
 - (a) Find the location of the center of mass (center of gravity) of the plate structure, measured from the pin rotation O.
 - (b) Calculate the reaction force at the support A.

[5 marks]

[8 marks]

(c) If the plate structure has only pin support O without another support at A,Find the moment at O to hold the plate structure at still.

[7 marks]

- Q5 A 3.5 kg slender bar AB rotates (clockwise direction) about the pivot rotation at O starting at vertical position with initial angular velocity of 2 rad/s as shown in Figure Q5. A soft spring with a constant of 200 N/m is attached at the initial position. The original length of the spring is 120mm.
 - (a) Determine the inertia of the slender bar about the point of rotation (point O).

[5 marks]

(b) Calculate the change of kinetic energy after the slender bar has rotated 90°.

[5 marks]

(c) Calculate the change of total potential energy after the slender bar has rotated 90°.

[5 marks]

(d) Find the angular velocity of the slender bar about the pivot of rotation O.

[5 marks]









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EMESTER / SESSION : SEMESTER 2 - 2010/11	PROGRAM : 2 BDD		
OURSE : DYNAMICS	COURSE CODE : BDA 20103 / BDA 2013		
DURSE : DYNAMICS $s = s_0 + v_0 t + \frac{1}{2} at^2$ $v = v_0 + at$ $v^2 = v_0^2 + 2as$ $\theta = \theta_0 + \omega t + \frac{1}{2} \alpha t^2$ $\omega = \omega_0 + \alpha t$ $\omega^2 = \omega_0^2 + 2\alpha s$ $v = v^r + v^\theta$ $v^\theta = r\omega$ $v^r = \dot{r}$ $a = a^r + a^\theta$ $a^r = \ddot{r} - \dot{\theta}^2 r$ $a^\theta = \ddot{\theta}r + 2\dot{\theta}\dot{r}$ $a = a^n + a^t$ $a^n = r\omega^2 = \frac{v^2}{r}$ $a^t = r\alpha$ $T_1 + U_{1\rightarrow 2} = T_2$ $T_1 + V_1 = T_2 + V_2$ $U = \Delta T + \Delta V_g + \Delta V_e$ $\Delta T = \frac{1}{2}m(v_2^2 - v_1^2) + \frac{1}{2}I_G(\omega_2^2 - \omega_1^2)$ $\Delta V_g = mg(h_2 - h_1)$ $\Delta V_e = \frac{1}{2}k(x_2^2 - x_1^2)$ $mv_1 + \sum_{l_1}^{l_2}Fdt = mv_2$	COURSE CODE : BDA 20103 / BDA 2013 $m_A(v_A)_1 + m_B(v_B)_1 = m_A(v_A)_2 + m_B(v_B)_2$ $I_G \omega_1 + m(v_G)_1 d_1 + \sum \int M_A dt = I_G \omega_2 + m(v_G)_2 d_2$ $e = -\frac{(v_B)_1^n - (v_A)_1^n}{(v_B)_1^n - (v_A)_1^n}$ $(v_A)_1^i = (v_A)_2^i$ $\sum M_G = I_G \alpha$ $\sum F = ma$ $v_P = v_P + v_{PlOxy}$ $v_P = (\vec{r})_{OXY} = \Omega \times \mathbf{r} + (\hat{\mathbf{r}})_{Oxy}$ $\mathbf{a}_P = \mathbf{a}_P + \mathbf{a}_{PlOxy} + \mathbf{a}_C$ $\mathbf{a}_P = \Omega \times (\Omega \times \mathbf{r}) + \dot{\Omega} \times \mathbf{r} + 2(\Omega \times (\hat{\mathbf{r}})_{Oxy}) + (\hat{\mathbf{r}})_{Oxy}$ $I = mk_G^2$ $I = \int_m r^2 dm$ $I_x = \frac{1}{18}mh^2$ $I_y = \frac{1}{18}m(b^2 + h^2)$ $I_{XX} = I_{YY} = \frac{1}{12}ml^2$ $I_{XX} = \frac{1}{12}m(A^2 + B^2)$ $I_{ZZ} = \frac{1}{12}m(A^2 + C^2)$		

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