

**UNIVERSITI TUN HUSSEIN ONN MALAYSIA**

**FINAL EXAMINATION  
SEMESTER II  
SESSION 2010/2011**

**COURSE NAME** : **MECHANICS OF MATERIALS II**  
**COURSE CODE** : **BDA 3033**  
**PROGRAMME** : **BACHELOR OF MECHANICAL  
ENGINEERING WITH HONOURS**  
**EXAMINATION DATE** : **APRIL / MAY 2011**  
**DURATION** : **3 HOURS**  
**INSTRUCTION** : **ANSWER FIVE (5) QUESTIONS  
OUT OF SIX (6) QUESTIONS**

**THIS QUESTION PAPER CONSISTS OF NINE (9) PAGES**

- S1** Keputusan terikan pada satu unsur dalam **RAJAH S1** memberikan nilai-nilai terikan utama seperti berikut;

$$\varepsilon_1 = 708\mu \quad \text{and} \quad \varepsilon_2 = -104\mu .$$

Dengan menggunakan kaedah bulatan Mohr atau kaedah kiraan, kirakan:

- (a) Terikan purata,  $\varepsilon_{avg}$
- (b) Terikan-terikan normal,  $\varepsilon_x$  dan  $\varepsilon_y$
- (c) Terikan ricih,  $\gamma_{xy}$
- (d) Terikan ricih maksimum,  $\gamma_{mak}$

(20 markah)

- S2** **RAJAH S2** menunjukkan sebatang rasuk AB yang menanggung beban teragih seragam 2.5 kN/m, satu beban tumpu 5 kN dan satu momen 4 kNm.

- (a) Dapatkan persamaan kecerunan dan persamaan lengkung dengan jarak  $x$  dari penyokong A.
- (b) Hitungkan anjakan pada titik tengah D.

Diberi :  $E=200\text{GPa}$ ,  $I=6.87 \times 10^{-6} \text{ m}^4$

(20 markah)

- S3** (a) Pertimbangkan tiang unggul sebagaimana dalam **RAJAH S3(a)** yang mempunyai kedua-dua hujung pin. Tunjukkan bahawa beban kritikal yang bertindak diberi oleh persamaan  $P_{cr} = \frac{\pi^2 EI}{L^2}$

(10 Markah)

- (b) **RAJAH S3(b)** menunjukkan satu struktur kekuda yang dibuat daripada bar keluli A-36 dengan keratan rentas berbentuk bulat. Anggota-anggota tersebut disokong oleh pin pada hujung-hujungnya. Sekiranya daya yang dikenakan adalah  $P= 50 \text{ kN}$ , tentukan diameter anggota AB dalam gandaan 5 mm yang terhampir bagi mengelak anggota tersebut daripada meleding. Diberi  $E=210 \text{ GPa}$  dan  $\sigma_Y=250 \text{ MPa}$ .

(10 Markah)

**S4** Satu struktur yang mempunyai dua rod AB dan BC mempunyai nilai EI yang sama telah dikimpal bersama pada B seperti yang ditunjukkan dalam RAJAH S4. Dengan menggunakan Toeri Castigliano, tentukan:

- (a) Anjakan menegak pada titik B  
 (b) Kecerunan pada titik B

Handwritten note: *nelaly*

(20 Markah)

**S5** (a) Jawab Ya atau Tidak

- (i) Tekanan silinder yang mengandungi cecair dibawah tekanan dan ketebalannya adalah tidak kecil ( $t \geq d/20$ ) di klasfikasikan sebagai silinder tebal.  
 (ii) Ketebalan silinder tebal adalah kecil dan boleh diabaikan.  
 (iii) Dalam silinder tebal, tegasan circumferential adalah kosong.  
 (iv) Dalam silinder tebal, tegasan jejari pada permukaan dalam sama dengan magnitude tekanan cecair.  
 (v) Dalam silinder tebal, variasi dalam tegasan jejari sekeliling merentasi ketebalan adalah teori Lamé.

(5 Markah)

(b) Sebuah silinder tebal dengan hujung tertutup yang diperbuat daripada aloi-Al mempunyai diameter dalam 200 mm dan diameter luar 800 mm. Silinder dikenakan tekanan dalaman sebanyak 150 MPa. Tentukan tegasan-tegasan utama maksimum dan minimum pada permukaan dalaman silinder. ( $E = 72 \text{ GPa}$ )

(15 Markah)

**S6** (a) Apakah itu bahan rapuh dan mulur? (4 Markah)

(b) Senaraikan dua teori kegagalan bagi bahan mulur dan rapuh. (6 Markah)

(c) Keadaan unsur tegasan yang ditunjukkan dalam RAJAH S6 berlaku pada sebuah mesin keluli iaitu  $\sigma_y = 325 \text{ MPa}$ . Dengan menggunakan criteria tegasan ricih maksimum, tentukan samada berlaku alahan sekiranya:

- (i)  $\sigma_o = 200 \text{ MPa}$ ,  
 (ii)  $\sigma_o = 240 \text{ MPa}$ ,  
 (iii)  $\sigma_o = 280 \text{ MPa}$ .  
 (iv) Jika tidak berlaku alahan, tentukan faktor keselamatan.

(10 Markah)

- Q1** Strains result of an element in **RAJAH S1** shows the principle strain values as below:-

$$\varepsilon_1 = 708\mu \quad \text{and} \quad \varepsilon_2 = -104\mu .$$

By using Mohr's circle or an analysis methods, calculate:

- (a) Average strain,  $\varepsilon_{\text{avg}}$
- (b) Normal strain,  $\varepsilon_x, \varepsilon_y$
- (c) Shear strain,  $\gamma_{xy}$
- (d) Maximum shear strain,  $\gamma_{\text{max}}$

(20 Marks)

- Q2** Beam AB supports a uniformly distributed load of 2.5 kN/m, one concentrated load of 5 kN and a moment of 4 kNm as shown in **RAJAH S2**.

- (a) Express the slope and deflection as function of the distance x from the support at A
- (b) Determine the deflection at the midpoint D

Use  $E = 200 \text{ GPa}$ , and  $I = 6.87 \times 10^{-6} \text{ m}^4$

(20 Marks)

- Q3** (a) Consider an ideal column as in **RAJAH S3(a)** having both ends pinned. Show that the critical load on the column is given by  $P_{cr} = \frac{\pi^2 EI}{L^2}$

(10 Marks)

- (b) The truss shown in **RAJAH S3(b)** is made from A-36 steel bars, each of which has a circular cross-section. The members are pin supported at their ends. If the applied load  $P = 50 \text{ kN}$ , determine the diameter of member AB to the nearest multiples of 5 mm that will prevent this member from buckling. Given  $E = 210 \text{ GPa}$  and  $\sigma_y = 250 \text{ MPa}$ .

(10 Marks)

- Q4** A structure has two rods AB and BC of the same flexural rigidity EI are welded together at B as shown in **RAJAH S4**. Using Castigliano's theorem, determine:

- (a) The horizontal deflection of point B
- (b) The slope of point B

(20 Marks)

- Q5** (a) Answer either Yes or No
- (i) Cylinder vessel, containing fluid under pressure and whose wall thickness is not small ( $t \geq d/20$ ) are classified as thick cylinder.
  - (ii) In thick cylinders the radial stress in the wall thickness is negligibly small.
  - (iii) In thick cylinders the circumferential stress is zero.
  - (iv) In a thick cylinder, the radial stress at inner surface is equal to the magnitude of the fluid pressure
  - (v) In thick cylinder the variation in the radial as well as circumferential stresses across the thickness is Lamé's theory.
- (5 Marks)
- (b) A closed-end thick cylinder walled is made of an Al-alloy has inside diameter of 200 mm and outside diameter of 800 mm. The cylinder is subjected to internal fluid pressure of 150 MPa. Determine the principal stresses and maximum shear stress at a point on the inside surface of the cylinder. ( $E = 72 \text{ GPa}$ )
- (15 Marks)
- Q6** (a) What is a ductile and brittle material? (4 Marks)
- (b) List of two theories of failure involved in ductile and brittle material. (6 Marks)
- (c) The state of plane stress shown in **RAJAH S6** occurs in a machine component made of a steel with  $\sigma_y = 325 \text{ MPa}$ . Using the maximum-shearing-stress criterion, determine whether yield occurs when:
- (i)  $\sigma_o = 200 \text{ MPa}$ ,
  - (ii)  $\sigma_o = 240 \text{ MPa}$ ,
  - (iii)  $\sigma_o = 280 \text{ MPa}$ .
  - (iv) If yield does not occur, determine the corresponding factor of safety.
- (10 Marks)

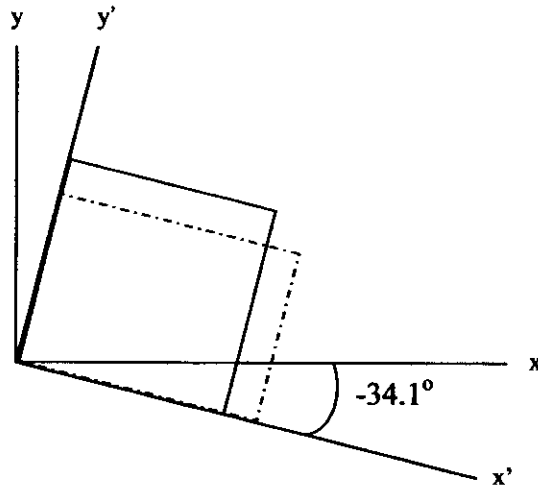
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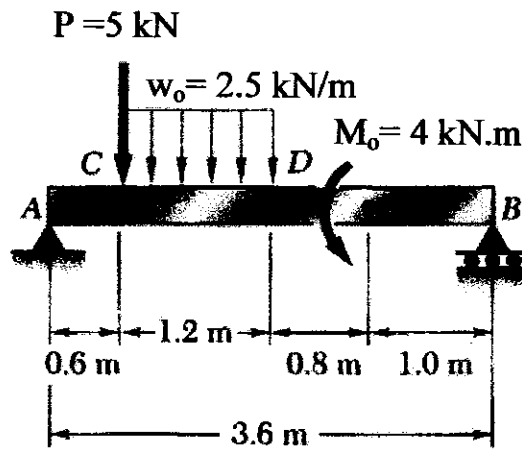
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**RAJAH S1**

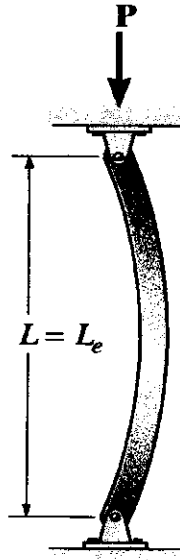


**RAJAH S2**

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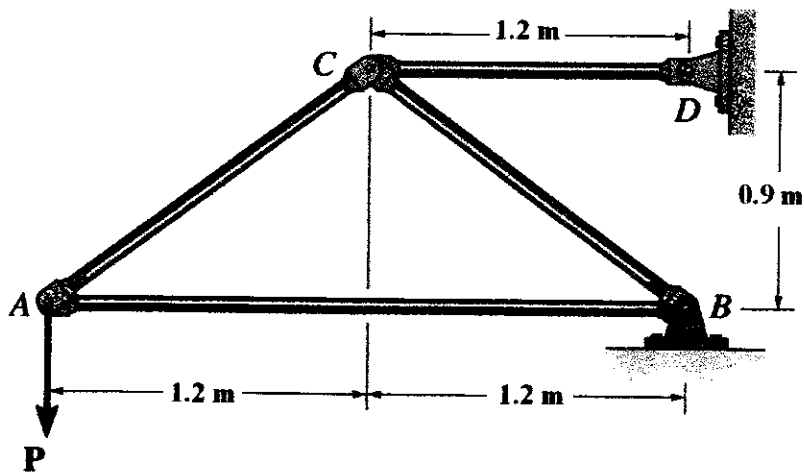
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Pinned ends

**RAJAH S3(a)**



**RAJAH S3(b)**

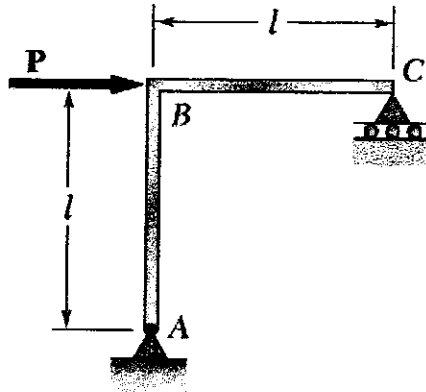
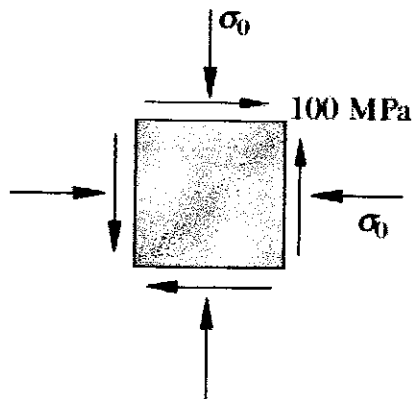
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**RAJAH S4****RAJAH S6**



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### Strain Transformation Equation

$$\varepsilon_{x'} = \frac{\varepsilon_x + \varepsilon_y}{2} + \frac{\varepsilon_x - \varepsilon_y}{2} \cos 2\theta + \frac{\gamma_{xy}}{2} \sin 2\theta$$

$$\frac{\gamma_{x'y'}}{2} = -\left(\frac{\varepsilon_x - \varepsilon_y}{2}\right) \sin 2\theta + \frac{\gamma_{xy}}{2} \cos 2\theta$$

$$\varepsilon_{1,2} = \frac{\varepsilon_x + \varepsilon_y}{2} \pm \sqrt{\left(\frac{\varepsilon_x - \varepsilon_y}{2}\right)^2 + \left(\frac{\gamma_{xy}}{2}\right)^2}$$

$$\tan 2\theta_p = \frac{\gamma_{xy}}{\varepsilon_x - \varepsilon_y}$$

$$\left(\frac{\gamma}{2}\right)_{\max} = \sqrt{\left(\frac{\varepsilon_x - \varepsilon_y}{2}\right)^2 + \left(\frac{\gamma_{xy}}{2}\right)^2}$$

$$\tan 2\theta_s = -\left(\frac{\varepsilon_x - \varepsilon_y}{\gamma_{xy}}\right)$$

### Stress Transformation Equation

$$\sigma_{x'} = \frac{\sigma_x + \sigma_y}{2} + \frac{\sigma_x - \sigma_y}{2} \cos 2\theta + \tau_{xy} \sin 2\theta$$

$$\tau_{x'y'} = -\left(\frac{\sigma_x - \sigma_y}{2}\right) \sin 2\theta + \tau_{xy} \cos 2\theta$$

$$\sigma_{1,2} = \frac{\sigma_x + \sigma_y}{2} \pm \sqrt{\left(\frac{\sigma_x - \sigma_y}{2}\right)^2 + (\tau_{xy})^2}$$

$$\tan 2\theta_p = \frac{\tau_{xy}}{(\sigma_x - \sigma_y)/2}$$

$$(\tau_{\text{in-plane}})_{\max} = \sqrt{\left(\frac{\sigma_x - \sigma_y}{2}\right)^2 + (\tau_{xy})^2}$$

$$\tan 2\theta_s = -\frac{(\sigma_x - \sigma_y)/2}{\tau_{xy}}$$

### Material-Property Relationship

$$E = \frac{\sigma}{\varepsilon}$$

$$G = \frac{\tau}{\gamma}$$

$$G = \frac{E}{2(1+\nu)}$$

$$\varepsilon_x = \frac{1}{E} [\sigma_x - \nu(\sigma_y + \sigma_z)]$$

### Buckling

$$P_{cr} = \frac{\pi^2 EI}{(KL)^2}$$

### Axial Load

$$\sigma = \frac{P}{A}$$

### Torsion

$$\tau = \frac{Tr}{J}$$

### Bending

$$\sigma = \frac{My}{I}$$

### Thick Cylinder

$$\sigma_H = A + \frac{B}{r^2}$$

$$\sigma_R = A - \frac{B}{r^2}$$

### Energy Methods

$$U_i = \frac{N^2 L}{2AE}$$

$$U_i = \int_0^L \frac{M^2}{EI} dx$$

$$U_i = \int_0^L \frac{T^2}{2GJ} dx$$

$$U_i = \int_0^L \frac{f_s V^2}{2GA} dx$$

### Failures Theories

#### Maximum-principal-stress theory

$$\sigma_1 = \sigma_Y$$

#### Maximum-principal-strain theory

$$\sigma_1 - \nu\sigma_2 = \sigma_Y$$

#### Maximum-shear-stress theory

$$\sigma_1 - \sigma_2 = \sigma_Y$$

#### Strain-energy theory

$$\sigma_1^2 + \sigma_2^2 - 2\nu\sigma_1\sigma_2 = \sigma_Y^2$$

#### Maximum-distortion-energy theory

$$\sigma_1^2 + \sigma_2^2 - \sigma_1\sigma_2 = \sigma_Y^2$$



**UNIVERSITI TUN HUSSEIN ONN MALAYSIA  
BORANG PENYEDIAAN KERTAS SOALAN PEPERIKSAAN AKHIR**

**BAHAGIAN A  
(Untuk Diisi Oleh Staf Akademik)**

Peperiksaan : Semester \* I II III Sesi 10 / 11

1.0 Nama 1. NASUHA B. SA'UDE 8443/0197287605  
(HURUF BESAR) (Samb. Tel/HP)

2. \_\_\_\_\_  
(HURUF BESAR) (Samb. Tel/HP)

3. \_\_\_\_\_  
(HURUF BESAR) (Samb. Tel/HP)

Kod Kursus : BDD4043  
(Nyatakan Kod Lain jika ada)  
Nama Kursus : RAPID PROTOTYPING  
Jangka masa Peperiksaan : 2 jam 30 minit  
Bilangan Pelajar : 88 orang

Bahasa :  Bahasa Malaysia  Bahasa Inggeris

*(Bagi soalan Bahasa Inggeris hendaklah disediakan terjemahannya dalam Bahasa Malaysia)*

2.0 Draf Kertas Soalan ini terdiri daripada:

- 1. Soalan Subjektif :    muka surat
- 2. Gambar rajah :    muka surat
- 3. Pelan :    muka surat
- 4. Carta :    muka surat
- 5. Soalan Objektif :    muka surat

3.0 Bahan rujukan dan keperluan-keperluan lain yang perlu disediakan oleh PPA :- Bilangan Dikehendaki

1. TIADA \_\_\_\_\_

2. \_\_\_\_\_

3. \_\_\_\_\_

4.0 Lain-lain arahan :

TIADA \_\_\_\_\_

NASUHA B. SA'UDE \_\_\_\_\_ 11/3/11  
Tandatangan dan Nama Rasmi Jabatan Kej. Pembuatan & Industri Tarikh

**BAHAGIAN B**  
(Untuk Dekan/Ketua Pusat)

5.0 Kertas soalan ini telah disemak dan ianya adalah diluluskan untuk dikeluarkan pada peperiksaan Semester \* I / II III Sesi 10 / 11

[Signature] \_\_\_\_\_ 31/3/2011  
Tanda tangan Dekan/Ketua Pusat Tarikh

**PROF. DR. SOLAMAN BIN HAJI HASAN**  
Dekan  
Fakulti Kejuruteraan Mekanikal & Pembuatan  
Universiti Tun Hussein Onn Malaysia  
86400 Parit Raja, Batu Pahat  
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